

VIBRATION ANALYSIS

HOW TO START A PREDICTIVE MAINTENANCE PROGRAM

Richard D. Hall

National Electrical Carbon Products

**Western Mining Electrical Association
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Types of Maintenance Systems

- * **Reactive**

- * **Proactive**

- **Preventive**

- **Predictive**

- **Root Cause Analysis**

There is a place for reactive maintenance

- * Non critical equipment**
- * Equipment that does not significantly affect production**
- * Equipment with adequate spares**
- * Inexpensive (disposable) equipment**
- * Machinery that does not impact safety**

Preventive Maintenance

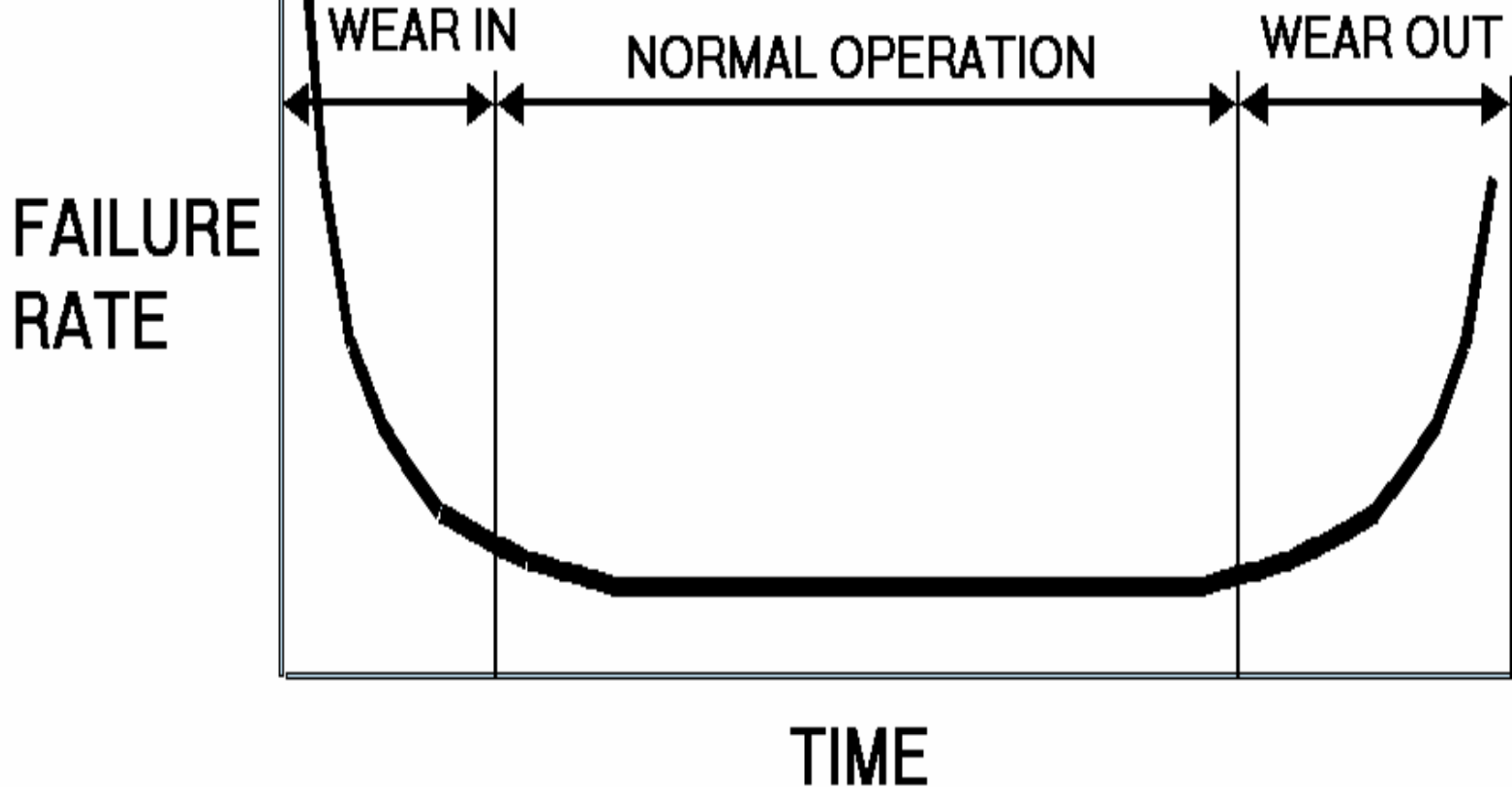
- * Routinely replace components based on machine history
- * Some maintenance is non-intrusive (oil leaks, structural cracks, excessive noise, heat, missing fasteners, worn brushes can be detected without disturbing machinery)
- * Replacing parts, however, can be hard on machinery and introduce new problems

A study carried out some years ago by two major US airline companies found that whenever intrusive maintenance is performed, problems are often introduced.

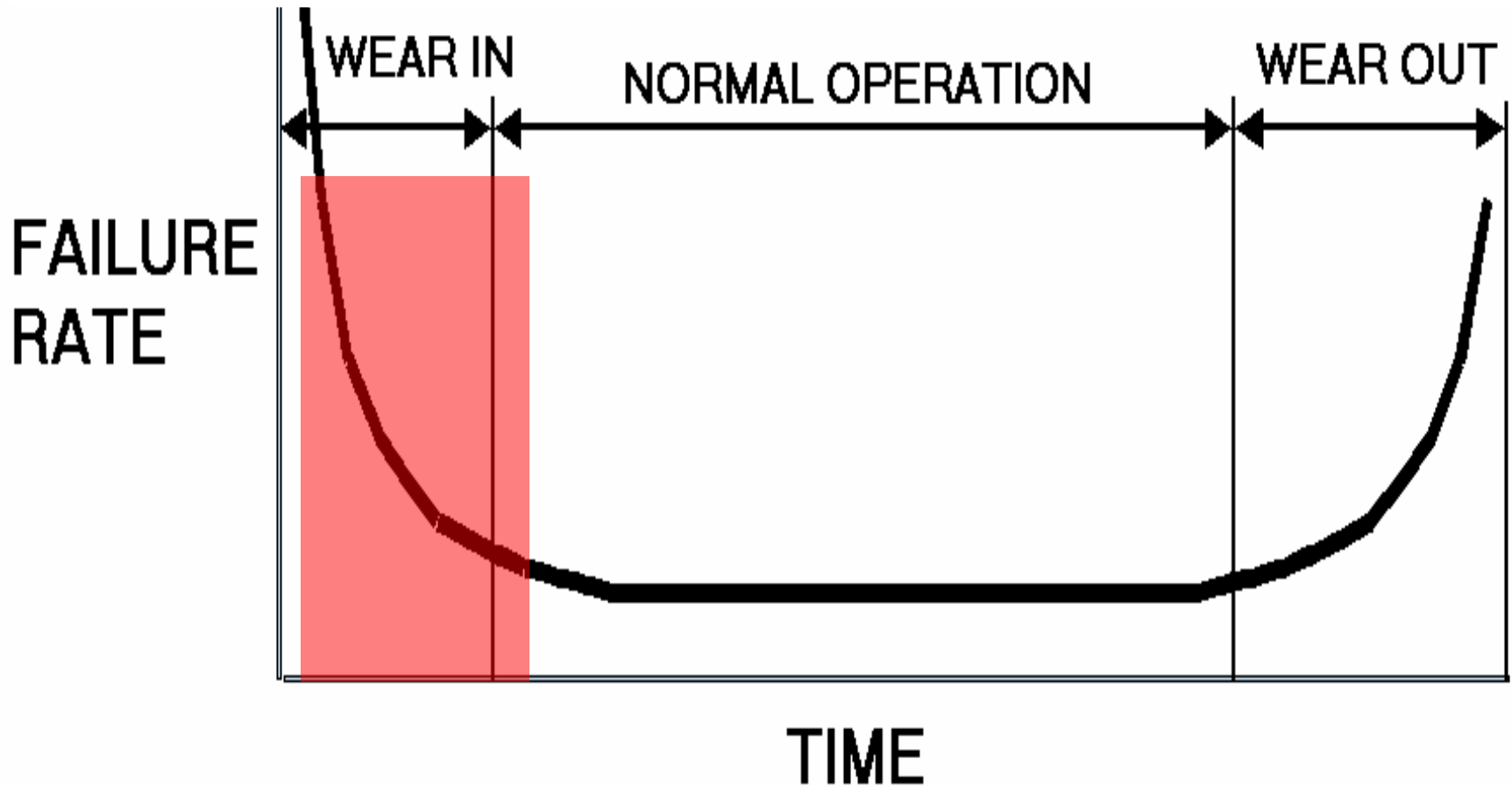
Contributing factors include -

- Defective parts**
- Misalignment**
- Unbalance**
- Incorrect assembly**

The result is called the
“BATH TUB” effect -



Every time you perform intrusive maintenance the machine goes through the “WEAR IN” period



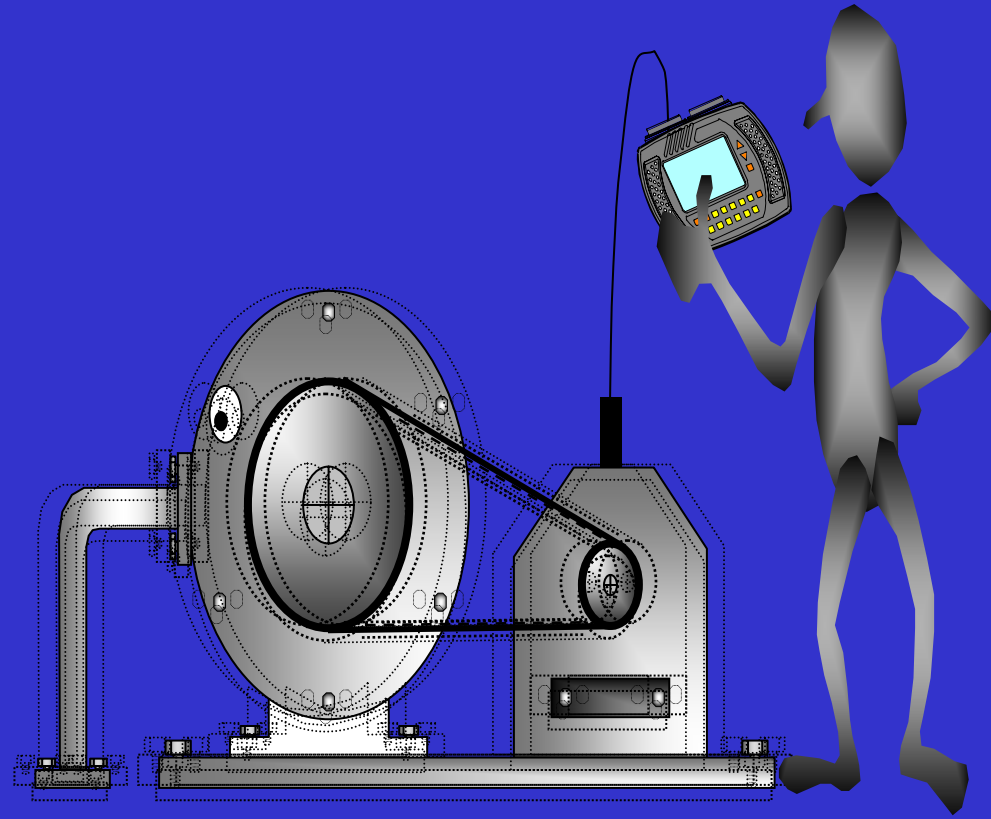
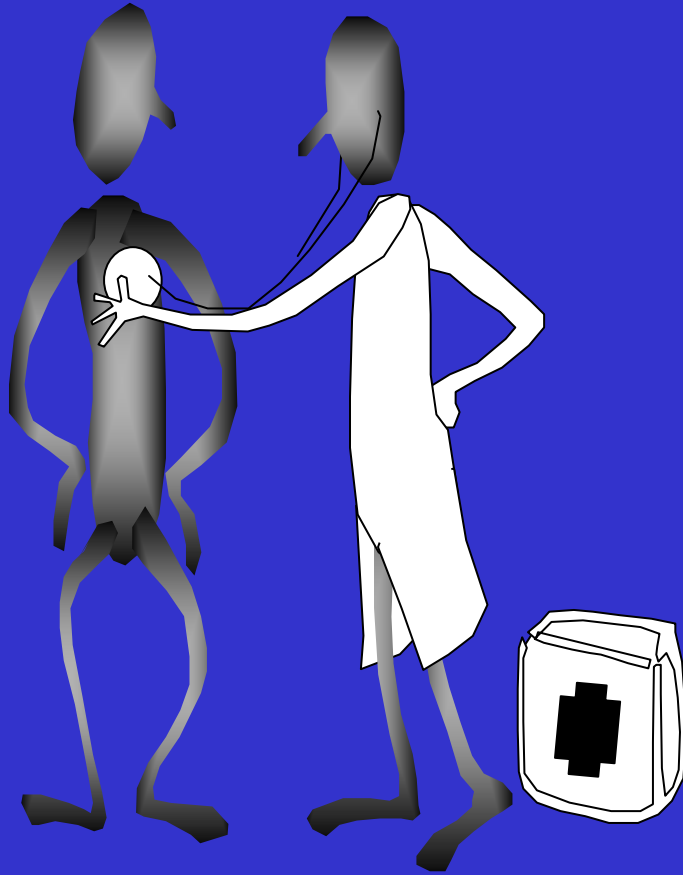
Lets consider the second
type of
PROACTIVE maintenance -

PREDICTIVE maintenance

PREDICTIVE maintenance involves regular monitoring to detect changes in a machine's operating condition

Predictive Maintenance might better be called “condition Monitoring”

**It is much like going to the doctor
for a regular checkup**



PROACTIVE MAINTENANCE makes
use of many technologies

Oil analysis

Ultra-Sonics

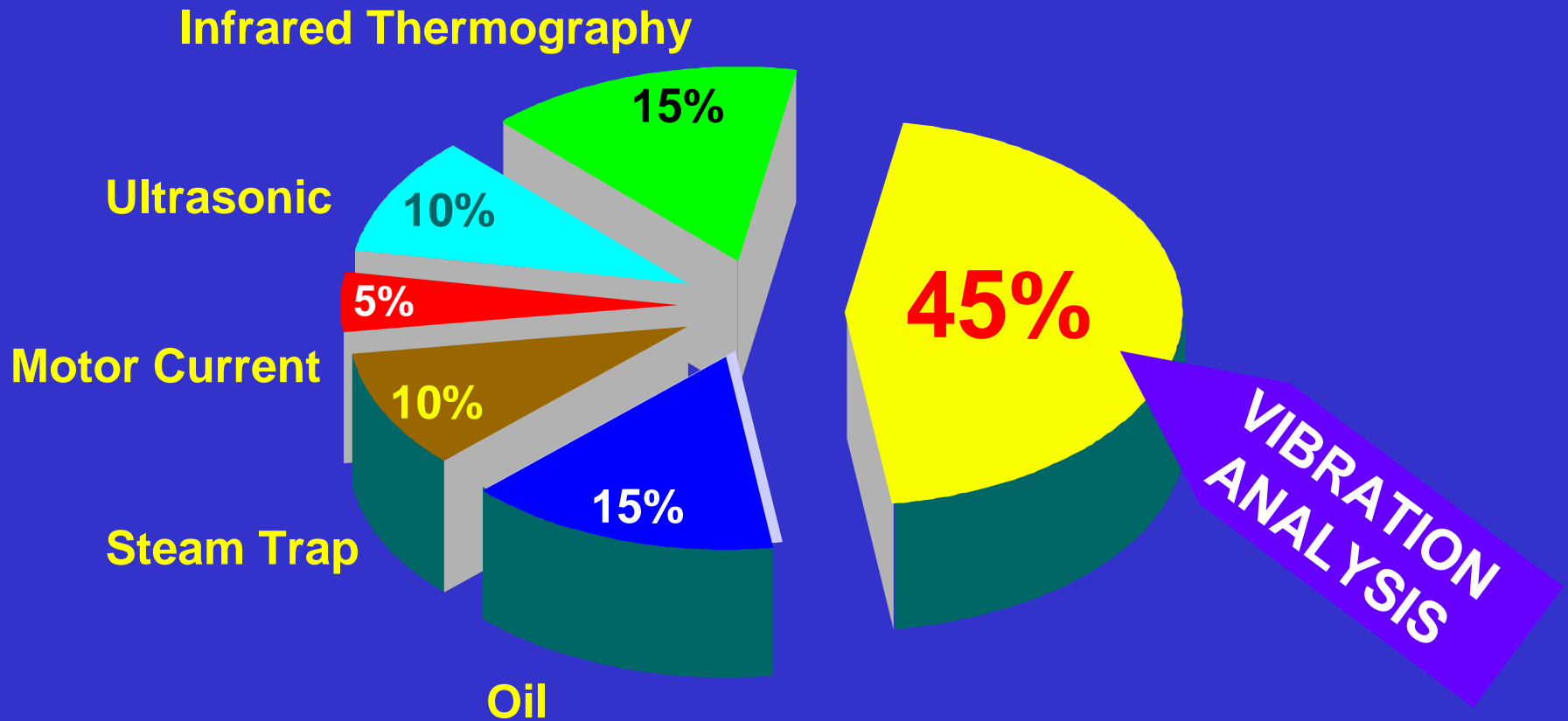
Infrared Thermography

Steam Trap

Motor Current

Vibration Analysis

One technology is distinctly MORE EFFECTIVE



Run to Failure

“If it ain’t broke, don’t fix it”

*** Ends up “broke”- often with expensive consequences**

Preventive Maintenance

“If it ain’t broke, fix it anyway”

*** Sometimes new problems are introduced (bathtub curve) and failures may actually increase. Unused good performance time is given up.**

Predictive Maintenance

If it is running fine and going to continue to do so, “do not disturb”. Only when it is “thinking about” having problems do you fix it.

* You get the most life out of the machine parts with the least disruption to production

Why Vibration Monitoring?

Vibration monitoring detects more problems than the next three predictive maintenance techniques combined.

Why Now?

Vibration monitoring equipment technology improvements have made the equipment much less expensive and easier to operate than in the past.



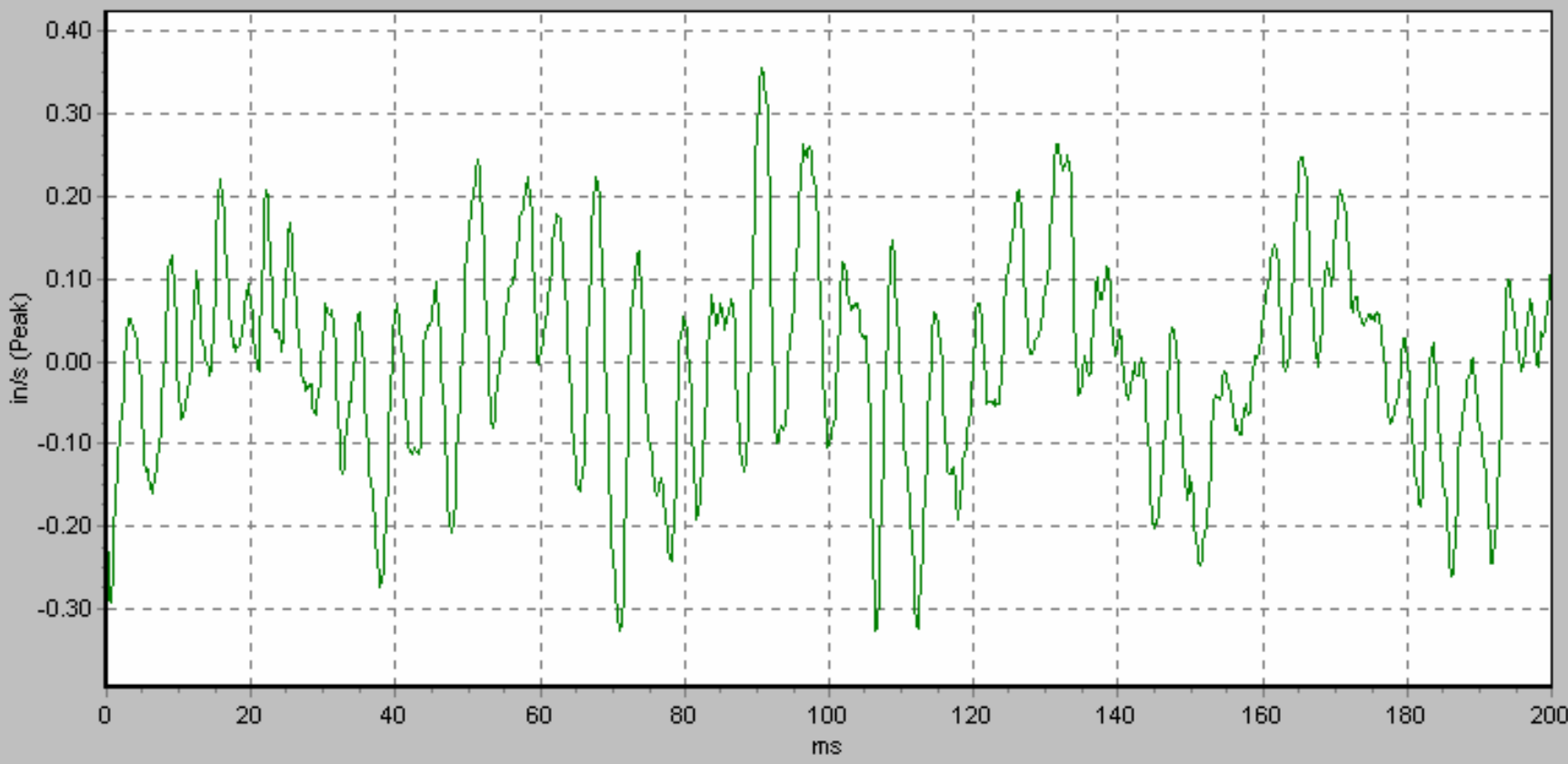


Plots Made with Vibration Analyzers

In the “Time Domain”

- * Plots vibration amplitude Vs time (Time Waveform)

MAS VB1



| Type | Plot | Machine | Point | Axis | Parameters | Recording start | O/All RMS | Samples | Company |
|------|------|-------------|----------------|------------|-----------------|--------------------------|---------------|---------|----------------|
| VB1 | █ | LINE GRIND1 | OPP DE BRG (1) | Horizontal | vel time 200 ms | 8 Nov 1999 (11:19:23 am) | 3.13 mm/s rms | 1024 | Trapper Mining |

In the “Time Domain”

- * Plots vibration amplitude Vs time (Time Waveform)

Typical problems detected

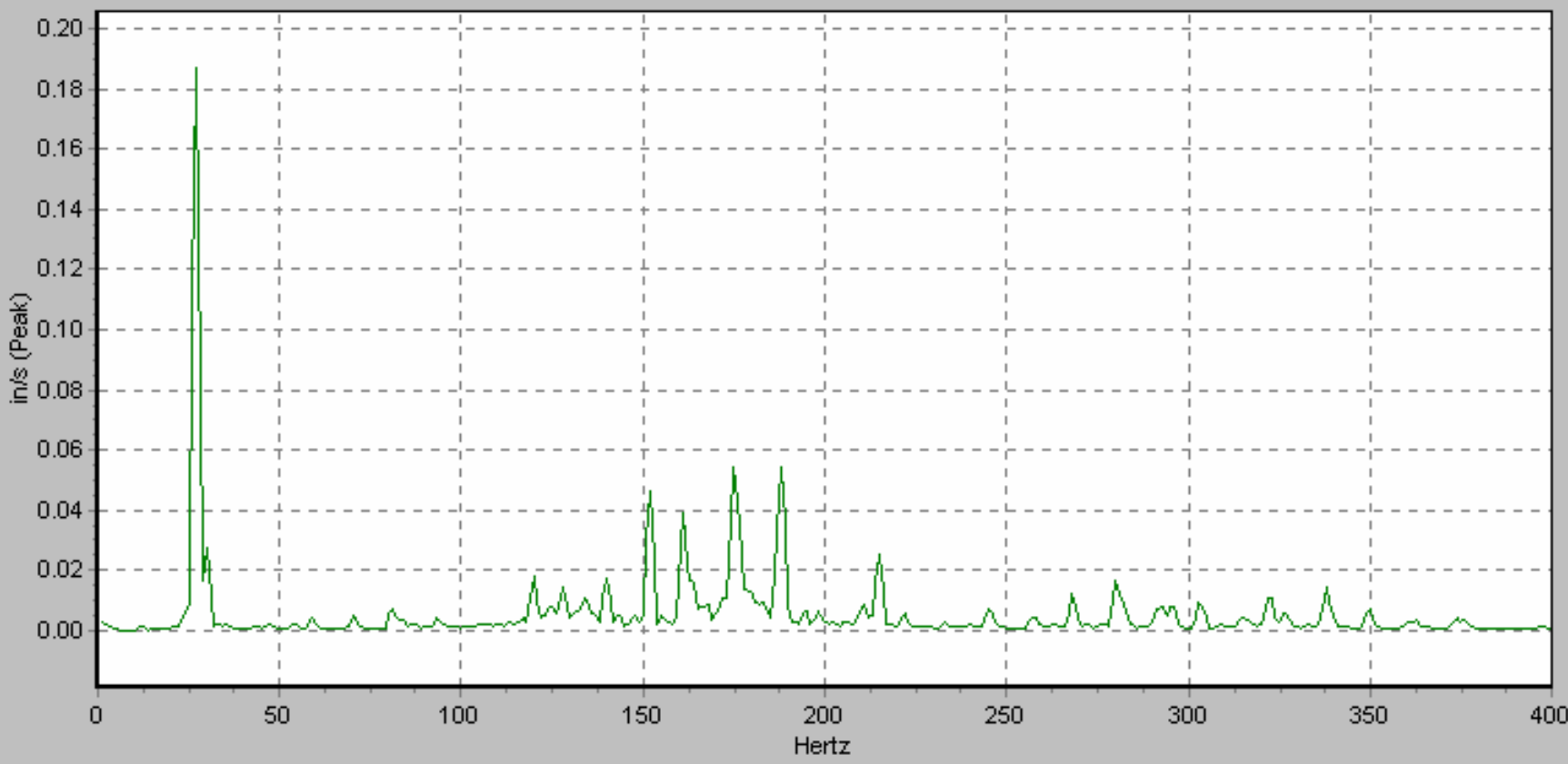
- * Broken gear teeth
- * AC motor problems
- * Amplitude modulation (interaction of two close frequencies)
- * Frequency modulation (frequency varies cyclically with time)

Plots Made with Vibration Analyzers

In the “Frequency Domain”

*** Plots vibration amplitude Vs frequency (frequency spectrum) Information is converted in the analyzer using a mathematical function called a Fast Fourier Transform or FFT**

MAS VB1



| Type | Plot | Machine | Point | Axis | Parameters | Recording start | O/All RMS | Spectrum size | FMax | Company |
|------|------|-------------|----------------|------------|-----------------|--------------------------|---------------|---------------|--------|----------------|
| VB1 | | LINE GRIND1 | OPP DE BRG (1) | Horizontal | vel freq 400 Hz | 8 Nov 1999 (11:21:54 am) | 4.28 mm/s rms | 400 lines | 400 Hz | Trapper Mining |

Plots Made with Vibration Analyzers

In the “Frequency Domain”

Frequency of the peaks indicate the cause of the problem

Amplitude of the peaks indicate the severity of the problem

In the “Frequency Domain”

Typical problems detected

- * **Unbalance**
- * **Misalignment**
- * **Mechanical looseness**
- * **DC motor problems (SCR drives)**
- * **Antifriction bearing problems**
- * **Gear problems**
- * **Pump and fan problems**
- * **Bent shaft**
- * **Soft foot**
- * **Oil whip, oil whirl or shaft rub (sleeve bearings)**

Setting up a Vibration Monitoring Program

Selecting Equipment to Monitor

List Your Machines

Selecting Equipment to Monitor

Categorize Your Machines

- * **Critical**
- * **Essential**
- * **Non- Critical**

Selecting Equipment to Monitor

Critical Machines

- * **Safety critical- catastrophic failure would endanger humans or the environment**
- * **Process critical- machines that must run 24 hours per day**
- * **These should definitely be in a predictive maintenance program**

Selecting Equipment to Monitor

Essential Machines

- * One of a kind with no spares
- * Machines with intermittent use, but must have guaranteed availability (like emergency generators, some compressors)
- * Machines where the cost to repair is extremely high
- * These should would be the next priority in a predictive maintenance program

Selecting Equipment to Monitor

Non-Essential Machines

- * Have little impact on production
- * Spare parts or replacement equipment is readily available
- * Repairs are not difficult or expensive
- * These would not be candidates for a predictive maintenance program

Selecting Equipment to Monitor

Within each category they should be further ranked by asking:

- * Is the machine fully spared, partially spared or not spared?
- * If spared, how long would it take for the spare to be put on line ?
- * To what level would plant production be reduced if this machine failed?
- * How much lead time is required for repair of this machine?
- * How costly is the repair/replacement of this machine?
- * Is the spare in good enough shape to continue production for a substantial length of time?
- * Does quality suffer due to poor performance or loss of this machine?

Selecting Equipment to Monitor

- * **Machines that have a chronic maintenance history/or recurring problems (these may require root cause analysis)**
- * **Machines with excessive replacement component or repair lead time**
- * **Machines that must be scheduled for repair far in advance**
- * **Machines under warranty and/or an insurance liability**



Potential Equipment to Monitor

Main MG Set

- * Rotating unbalance
- * Resonant frequencies
- * Electrical faults - generators
- * Electrical faults- synchronous motor
- * Antifriction thrust bearings
- * Coupling eccentricity

Potential Equipment to Monitor

Swing MG Set

- * Rotating unbalance
- * Resonant frequencies
- * Electrical faults - generators
- * Electrical faults- induction motor
- * Ball bearings
- * Coupling eccentricity

Potential Equipment to Monitor

Motion drive motors

- * Rotating unbalance
- * Resonant frequencies
- * Electrical faults
- * Misalignment
- * Antifriction bearings
- * Soft foot mounting (improper shimming)
- * Blower problems
- * Brakes

Potential Equipment to Monitor

Gear boxes

- * Rotating unbalance
- * Resonant frequencies
- * Gear teeth problems
- * Misalignment
- * Antifriction bearings
- * Eccentric gears or bent shafts

Potential Equipment to Monitor

Since the motion drives are more complex, we will investigate the swing drive as an example

Potential Equipment to Monitor

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- * **Motors**
- * **Bearings**
- * **Gear mesh**
- * **Alignment**

Six Steps to Analyzing Vibration

- 1. Familiarize yourself with the equipment. What is inside?**

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- 5. Identify other vibrations present**

Six Steps to Analyzing Vibration

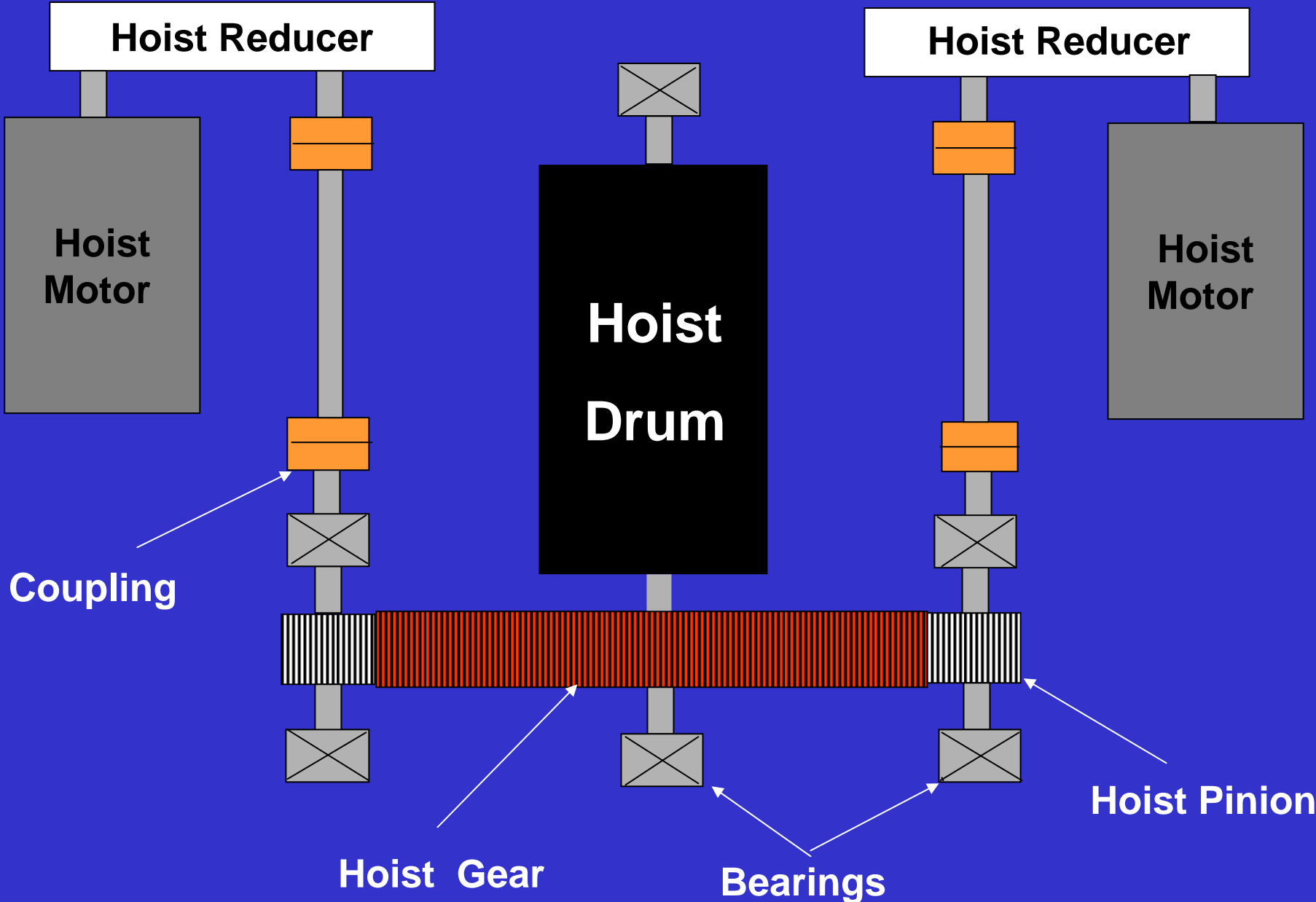
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- 6. Evaluate the equipment and make a recommendation**

**Familiarize yourself with the
equipment (Page 752 Dragline)**

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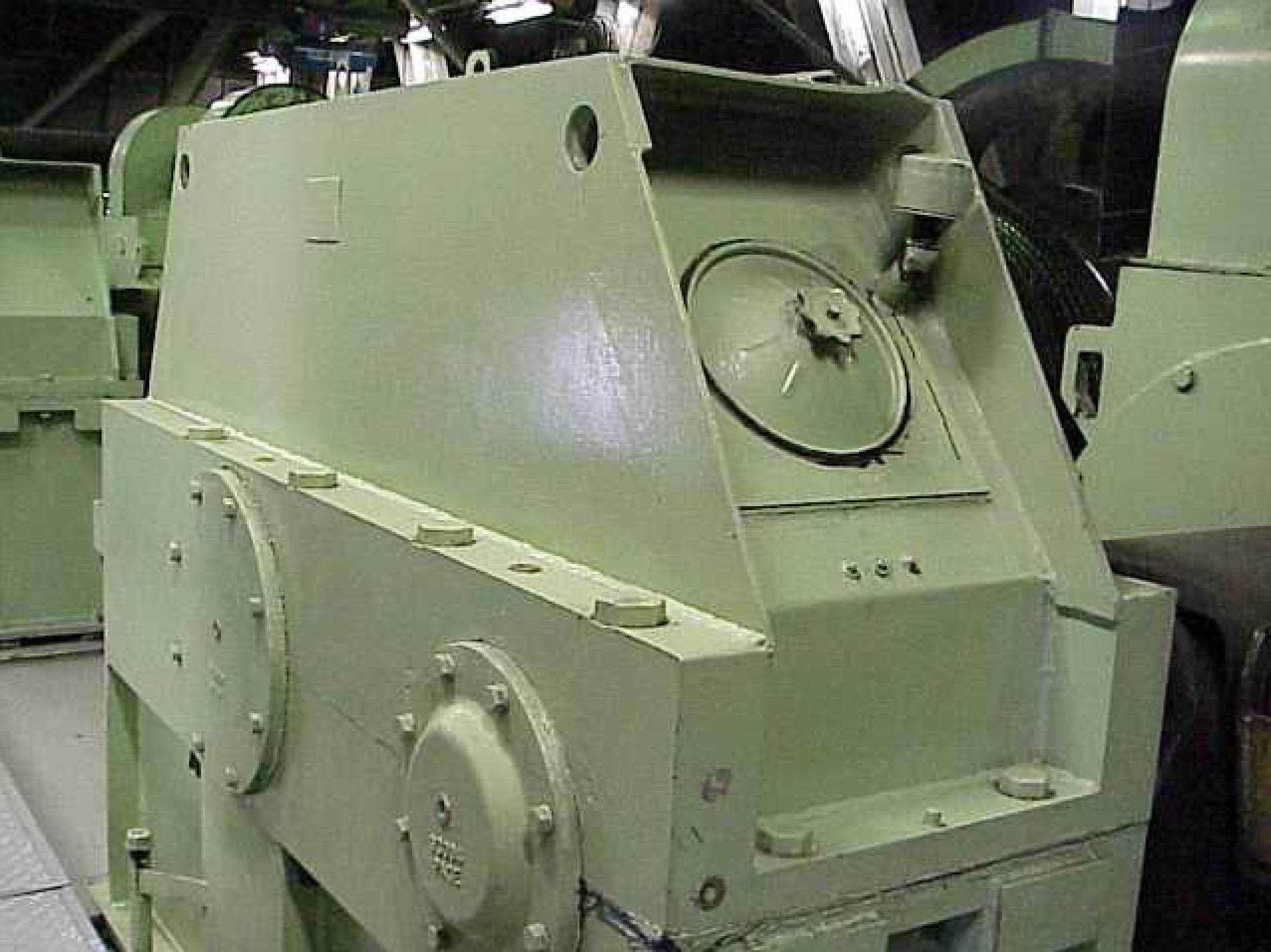
- a. Reviewed P&H (Page) drawings of machinery layout**

Hoist Motion

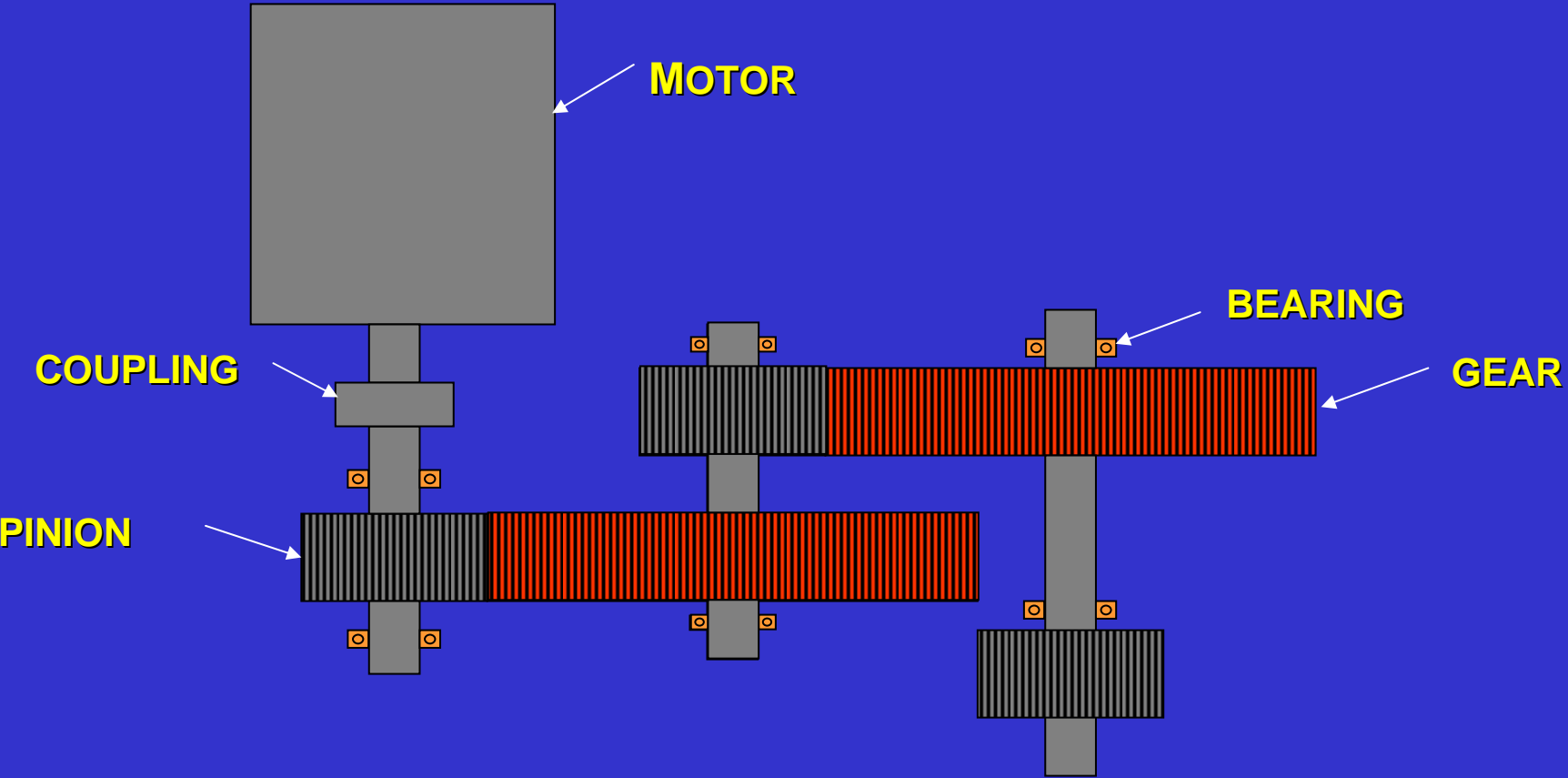


Familiarize yourself with the equipment (Page 752 Dragline)

- a. Reviewed P&H (Page) drawings of machinery layout**
- b. Reviewed OEM drawings of gearboxes**



Swing Motion



Familiarize yourself with the equipment (Page 752 Dragline)

- a. Reviewed P&H (Page) drawings of machinery layout**
- b. Reviewed OEM drawings of gearboxes**
- c. Gathered part numbers of gears, shafts, bearings etc. from OEM drawings**

Familiarize yourself with the equipment (Page 752 Dragline)

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- b. Reviewed OEM drawings of gearboxes**
- c. Gathered part numbers of gears, shafts, bearings etc. from OEM drawings**
- d. Identified bearing manufacturers numbers (mine purchasing, GE, P&H engineering)**

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- a. Reviewed P&H (Page) drawings of machinery layout**
- b. Reviewed OEM drawings of gearboxes**
- c. Gathered part numbers of gears, shafts, bearings etc. from OEM drawings**
- d. Identified bearing manufacturers numbers (mine purchasing, GE, P&H engineering)**
- e. Determined the number of teeth on all gears**
Inspect spare or used gears at the mine



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5054-0012

5054-0011







Familiarize yourself with the equipment (Page 752 Dragline)

a. Reviewed P&H (Page) drawings of machinery layout

2

b. Reviewed OEM drawings of gearboxes

c. Gathered part numbers of gears, shafts, bearings etc. from OEM drawings

d. Identified bearing manufacturers numbers (mine purchasing, GE, P&H engineering)

e. Determined the number of teeth on all gears

Inspected spare or used gears at the mine

Obtained information from the OEM

Familiarize yourself with the equipment (Page 752 Dragline)

f. Determined bearing fault frequencies (BPFI, BPFO, BSF, FTF)

The bearing manufacturers know this and it is becoming more available, but is not universally, easily available today

- * “Bearing Expert” in the analyzer software (Timken)

 - Currently contains 1,000,000 bearing numbers with data on approximately 120,000

- * One bearing manufacturer sent diskettes with their bearing information (Torrington)

- * NTN said info soon on web site. Obtained by phone

Bearing Fault Frequencies- (e.g. Torrington HM256810CD (Cup))

BPFI- Ball Pass Frequency Inner race (e.g.. 19.441)

BPFO- Ball Pass Frequency Outer race (e.g. 16.559)

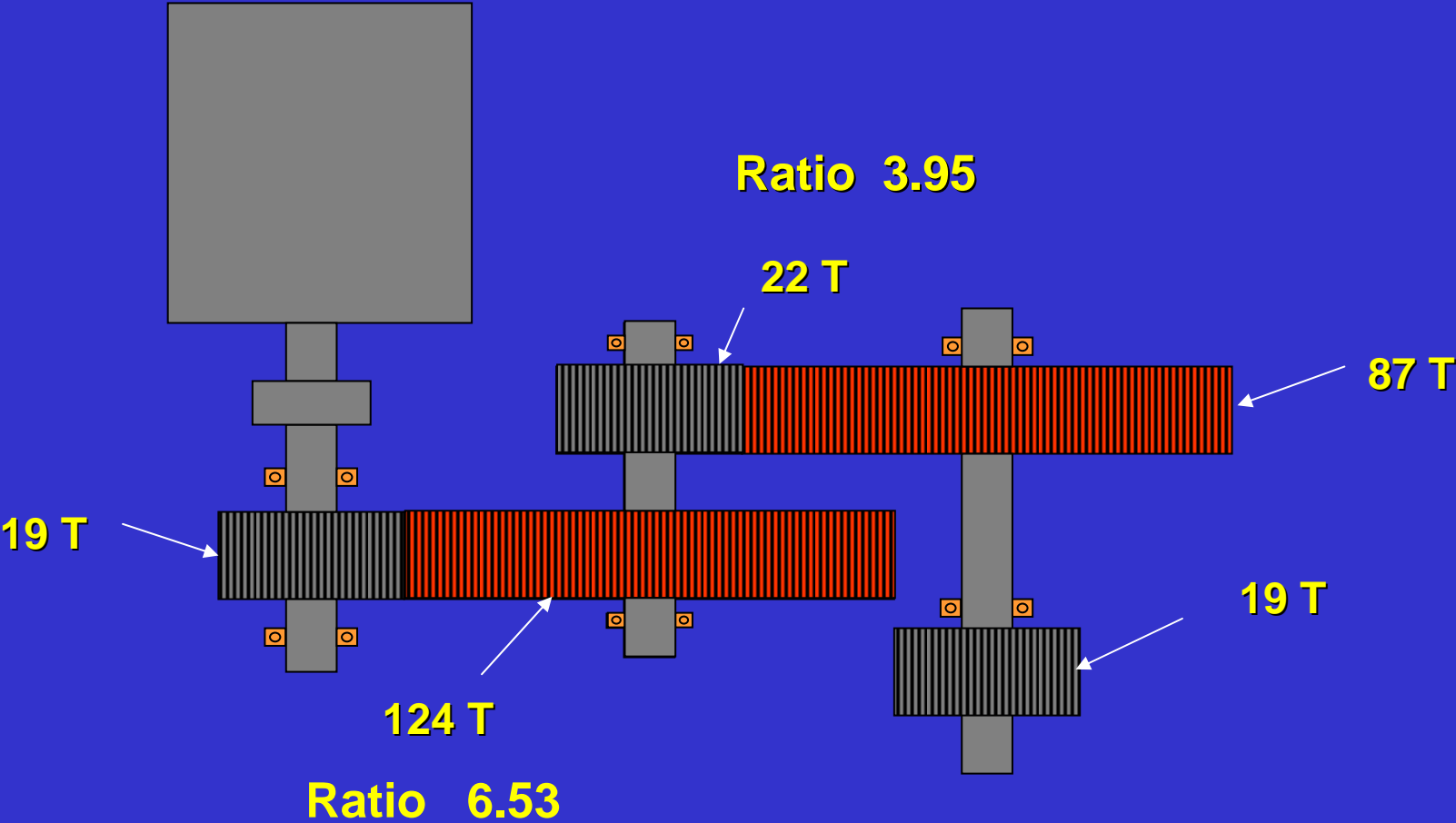
BSF- Ball Spin Frequency (e.g. 6.075)

FTF- Fundamental Train Frequency -cage rotation (e.g. 0.46)

Six Steps to Analyzing Vibration

- 1. Familiarize yourself with the equipment. What is inside?**
- 2. Calculate the important speeds and frequencies**

Swing Motion



Calculate Important Speeds and Frequencies (Swing Motor-Pin.Sft.)

| Item | Freq | Speed | Mult. | Freq. (kCPM) | Freq. (Hz) |
|-------------|----------|-------|--------|--------------|------------|
| Blo Mtr. | 1X | 1750 | 1 | 1.75 | 29.17 |
| Mtr/Pin Sft | 1X | 1000 | 1 | 1 | 16.67 |
| S. Mtr. | BPFI | 1000 | 8.3 | 8.3 | 138.33 |
| S. Mtr. | BPFO | 1000 | 5.7 | 5.7 | 95.00 |
| S. Mtr. | BSF | 1000 | 2.6 | 2.6 | 43.33 |
| S.Mtr. | FTF | 1000 | 0.41 | 0.4 | 6.83 |
| S. Mtr | Slot | 1000 | 58 | 58.0 | 966.67 |
| Pin Sft. | BPFI | 1000 | 17.296 | 17.3 | 288.27 |
| Pin Sft. | BPFO | 1000 | 14.704 | 14.7 | 245.07 |
| Pin Sft. | BSF | 1000 | 5.966 | 6.0 | 99.43 |
| Pin Sft. | FTF | 1000 | 0.459 | 0.5 | 7.65 |
| # Pin Teeth | 1st Mesh | 1000 | 19 | 19 | 316.67 |

Calculate Important Speeds and Frequencies (First Reduction Shaft)

Gear Ratio 6.53

Pinion Shaft 1000 RPM

| Item | Freq | Speed | Mult. | Freq. (kCPM) | Freq. (Hz) |
|--------------------|----------|--------|--------|--------------|------------|
| # Gear Teeth | 1st Mesh | 153.23 | 124 | 19.00 | 316.67 |
| 1st Red. Sft. | 1X | 153.23 | 1 | 0.15 | 2.55 |
| 1st Red. Sft. Brg. | BPFI | 153.23 | 13.356 | 2.05 | 34.11 |
| 1st Red. Sft. Brg. | BPFO | 153.23 | 10.644 | 1.63 | 27.18 |
| 1st Red. Sft. Brg. | BSF | 153.23 | 4.286 | 0.66 | 10.95 |
| 1st Red. Sft. Brg. | FTF | 153.23 | 0.444 | 0.07 | 1.13 |
| # Pin Teeth | 2nd Msh | 153.23 | 22 | 3.37 | 56.18 |

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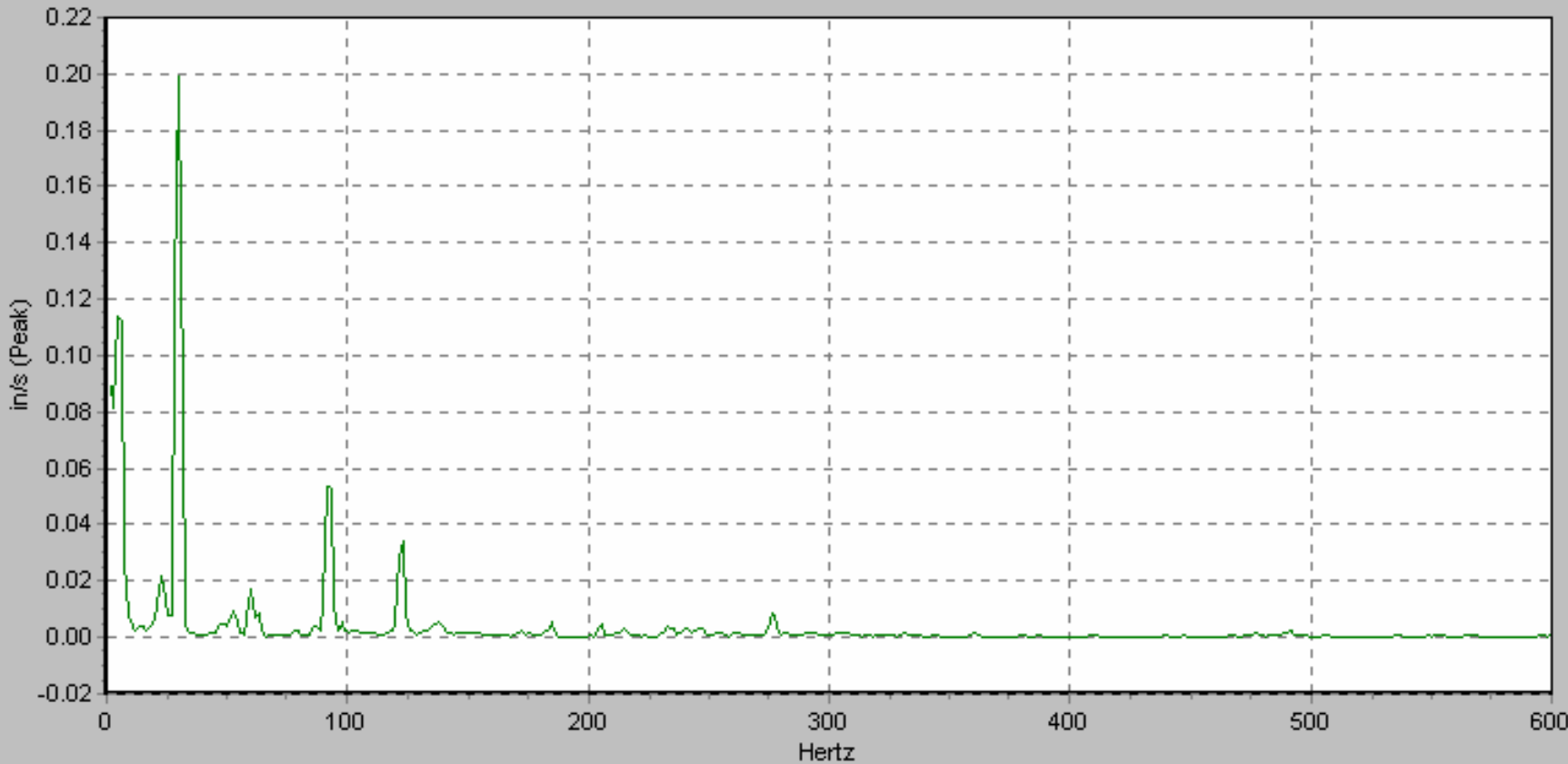


Info

Values

Plots

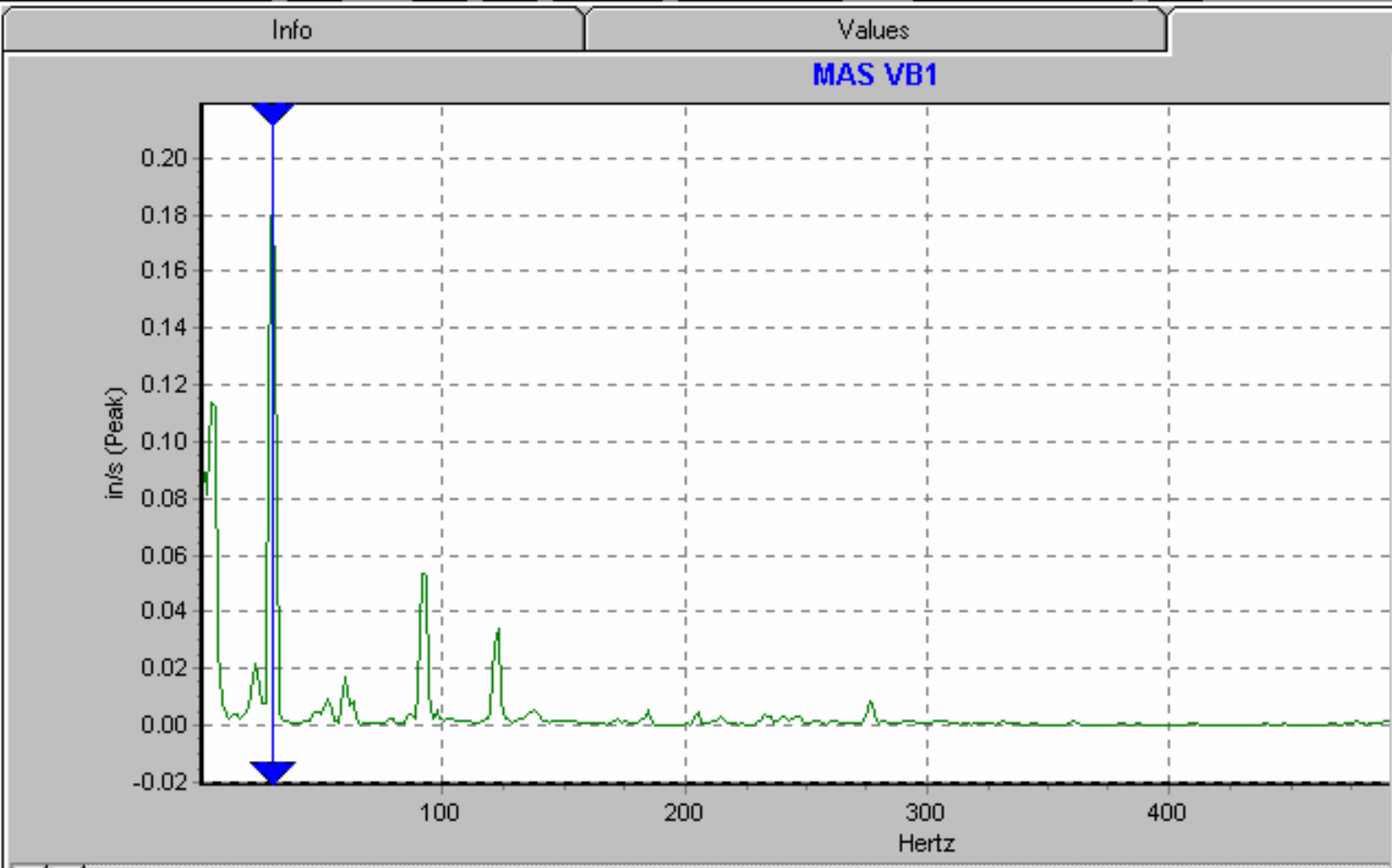
MAS VB1



| Type | Plot | Machine | Point | Axis | Parameters | Recording start | O/All RMS | Spectrum size | FMax | Company |
|------|------|---------|------------|------------|-----------------|---------------------------|----------------|---------------|--------|---------|
| VB1 | █ | SG3 | CE BRG (1) | Horizontal | vel freq 600 Hz | 12 Aug 1999 (12:42:10 pm) | 0.206 in/s rms | 400 lines | 600 Hz | |

C:\MAS\WIBRAT~1\TRAPPER.VB1

1 record selected



Plots

Cursors & Trends

cursor 1

Peak Search ◀ ▶

Hz in/s peak

cursor 2

Peak Search ◀ ▶

Hz in/s peak

difference

Hz in/s peak

ratio

threshold

in/s peak

| Type | Plot | Machine | Point | Axis | Parameters | Recording start | O/All RMS | Spectrum size | FMax | Compa |
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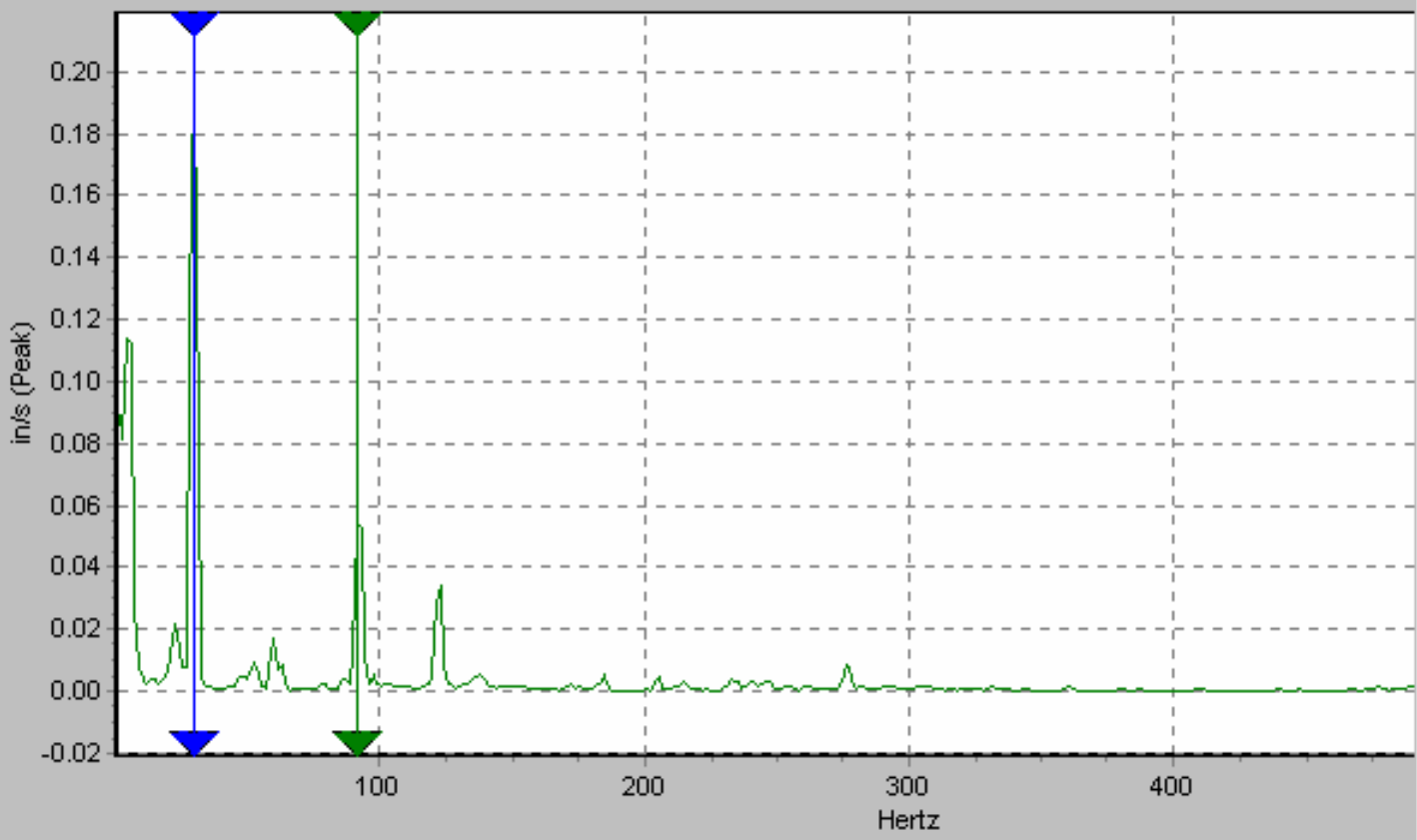
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Info

Values

Plots

MAS VB1



Cursors & Trends

cursor 1

Peak Search
 Hz in/s peak

cursor 2

Peak Search
 Hz in/s peak

difference

Hz in/s peak

ratio

threshold

in/s peak

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Vibration Measurement- My Current Thoughts

- * MG sets should be idling to avoid introducing vibration not related to the sets. Sets are constant speed, so taking measurements is easy.
- * Swing motion should be run steadily at top speed as this is the easiest way to repeat speed on later measurements. Possibly run both directions in case one side of gear teeth only has problems.
- * Hoist and drag. Run at top speed, empty bucket with machine not swinging. Limited time at top speed may be a challenge in getting readings.
- * Bearings- Some applications have a variety of bearing vendors approved (e.g. Motors and generators). Fault frequencies may vary by vendor. When repairs are made obtain info on bearing vendor and bearing number.



Special Thanks To:

Trapper Mining Inc.

- Terry Wooten
- Ted Crook

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- Harvey Kallenberger
- Paul Todd

GE

- Dennis Buto
- Millis Parshall



National Electrical Carbon Inc.
P.O. Box 1056
Greenville, SC 29602

Rich Hall

Phone 864-458-7700 Ext. 106

Fax 864-987-0449

e-mail: rhall@nationalelectrical.com

